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GUST RESPONSE OF ROTARY WING
AND ITS ALLEVIATION

by

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ABSTRACT

A simple feedback system to alleviate the gust response of the helicopter rotor is analyzed. The system is comprised of sensors each of which can detect the flapping motion of the respective blade and of actuators which can control the pitch angle of the blade following to the signal shaped adequately to the gust alleviation. The validity of this system is demonstrated by applying the theoretical calculation based on the local momentum theory to the gust responses of the complete dynamic system of a helicopter penetrating into a sinusoidal gust and a step gust.

1. INTRODUCTION

The helicopter has the potential to fly close to the ground over which the motion of atmosphere may be thought of turbulent flow. In order to avoid unfavorable dynamic characteristics in flying and riding qualities and structural vibrations caused by such gusty wind, it is required to analyze the gust response of the rotary wing and to study how to alleviate them.

The unsteady characteristics of the gust response of helicopter rotor were made clear in Ref. 1 by applying the local momentum theory (LMT)². The theory may also be applicable to the study of maneuver response of the helicopter without essential change of the computer program.

There have been proposed several ways to reduce the vibration of the helicopter by introducing the active pitch control corresponding to the change of load at the hub and blades³⁻⁷. Modern technology in the aircraft control engineering enables us to utilize advanced actuators which can operate the pitch control link of individual rotor blade in response to the control or command signal with high frequency.

Thus more free consideration on the design of feedback control system is possible for the gust alleviation of the helicopter rotor. By referring to the past results^{1,8,9} the present paper is to present a simple proposal for the control guidance of the gust alleviation at the initial stage in this field.

2. BLADE FLAPPING MOTION

The irregular flapping motion of the rotor blade must be a good indication for the existence of the gust along the flight course. Let us consider first the most simple blade-flapping motion of the rotor under the following assumptions: (i) a set of blades is made rigid and uniform, (ii) the gust is, by referring to Fig. 1, sinusoidal or

$$\begin{aligned} w_G(X,t) &= w_{G_0} \sin\{\kappa(X_0 - X + V_G t)\} \\ &= w_{G_0} \sin\{\kappa(X_0 - X) + \omega_G t\}, \end{aligned} \quad (1)$$

in which (iii) the angular frequency ω_G is smaller than that of the rotor speed or $\omega_G/\Omega \ll 1$, (iv) the blade pitch is given by the first harmonics such as

$$\theta = \theta_0 + \theta_t(x-3/4) + \theta_{1c} \cos\psi + \theta_{1s} \sin\psi, \quad (2)$$

and (v) the induced velocity is a kind of funnel shape given by

$$v = \bar{v} \{1 - (2/3)K_0 + x(K_0 + K_{1c} \cos\psi + K_{1s} \sin\psi)\} \quad (3)$$

where the meaning of all symbols have been listed in the NOMENCLATURE.

Then the equation of flapping motion of the blade is given by

$$\begin{aligned} & \ddot{\beta}/\Omega^2 + A(\psi)\dot{\beta}/\Omega + B(\psi)\beta \\ & = C(\psi) + \int_{x_\beta}^B \frac{1}{x^2} \rho a c R^3 \Omega^2 (x + \mu \sin\psi) (w_G/R\Omega) R(x - x_\beta) dx \quad (4) \end{aligned}$$

where coefficients $A(\psi)$, $B(\psi)$, and $C(\psi)$ are explained in APPENDIX A and where the final forced term in the right hand side of the above equation is given in APPENDIX B1).

By expanding the flapping angle β and the feathering angle θ in the series of first harmonics of the azimuth angle and by dividing them into non-perturbed and gust-perturbed terms,

$$\begin{aligned} \beta(t) &= \beta_n(t) + \Delta\beta(t) \\ &= \beta_{0n}(t) + \Delta\beta_0(t) + \{\beta_{n,1c}(t) + \Delta\beta_{1c}(t)\} \cos\psi + \{\beta_{n,1s}(t) + \Delta\beta_{1s}(t)\} \sin\psi \\ \theta(t) &= \theta_n(t) + \Delta\theta(t) \\ &= \theta_{0n}(t) + \Delta\theta_0(t) + \{\theta_{n,1c}(t) + \Delta\theta_{1c}(t)\} \cos\psi + \{\theta_{n,1s}(t) + \Delta\theta_{1s}(t)\} \sin\psi \end{aligned} \quad (5)$$

equation (4) yields the following two sets of equations of perturbed quantities which are mutually independent:

$$\begin{aligned} & \Delta\ddot{\beta}_0/\Omega^2 + K_\beta \Delta\dot{\beta}_0/\Omega + (1 + K_\beta) \Delta\beta_0 \\ & = (K_1 + \frac{1}{2}K_3\mu^2) \Delta\theta_0 + (w_{G_0}/R\Omega) (K_2 - \frac{1}{4}k^2K_4) \sin(\omega_G t) \quad (6) \end{aligned}$$

and

$$\begin{aligned} & \Delta\ddot{\beta}/\Omega^2 + (K_\beta - 2i) \Delta\dot{\beta}/\Omega + (K_\beta - iK_\beta) \Delta\beta \\ & = (K_1 + \frac{1}{2}K_3\mu^2) \Delta\theta + (w_{G_0}/R\Omega) \{i(\frac{1}{8}k^3K_5 - kK_1) \cos(\omega_G t) \\ & \quad + (\frac{1}{8}k^2K_1\mu - K_3\mu) \sin(\omega_G t)\}, \quad (7) \end{aligned}$$

where

$$\begin{aligned}\beta &= -\beta_{1s} + i\beta_{1c} \\ \theta &= -\theta_{1s} + i\theta_{1c}.\end{aligned}\quad (8)$$

It is, then, obvious that the first and the second equations, (6) and (7), are related to the coning angle and the complex tilt angle of a rigid cone which is assumed to be configured by the first harmonic motion of the one blade¹⁰.

3. FEEDBACK SYSTEM

Now let us consider the following simple feedback system:

$$\begin{aligned}\Delta\theta_0 &= \{b_\beta \Delta\beta_0 + b_\beta \dot{\Delta\beta}_0 / \Omega + b_\beta \ddot{\Delta\beta}_0 / \Omega^2\} / (K_1 + \frac{1}{2}K_3\mu^2) \\ \Delta\theta &= \{c_\beta \Delta\beta + c_\beta \dot{\Delta\beta} / \Omega + c_\beta \ddot{\Delta\beta} / \Omega^2\} / (K_1 + \frac{1}{2}K_3\mu^2).\end{aligned}\quad (9)$$

where any kind of time lag of the actuator has been out of consideration.. Then, the characteristic equations for equations (6) and (7) are respectively given by

$$(1 - b_\beta)(s/\Omega)^2 + (K_\beta - b_\beta)(s/\Omega) + (1 + K_\beta - b_\beta) = 0 \quad (10)$$

$$(1 - c_\beta)(s/\Omega)^2 + (K_\beta - 2i - c_\beta)(s/\Omega) + (K_\beta - iK_\beta - c_\beta) = 0 \quad (11)$$

Fig. 2 a, b, and c show the root loci of equation (10) for systems in which individual gain is feedback independently. The stable region of this feedback systems are respectively given by

$$\begin{aligned}(a) \text{ for } b_\beta &: b_\beta < K_\beta + 1 \\ (b) \text{ for } b_\beta &: b_\beta < K_\beta \\ (c) \text{ for } b_\beta &: b_\beta \leq 1.\end{aligned}\quad (12)$$

The undamped natural frequency and the damping ratio of equation (10) can be given by

$$\begin{aligned}\omega_n &= \sqrt{(1 + K_\beta - b_\beta) / (1 - b_\beta)} \\ \xi &= \{(K_\beta - b_\beta) / 2\} / \sqrt{(1 + K_\beta - b_\beta)(1 - b_\beta)}.\end{aligned}\quad (13)$$

Similarly, Fig. 3 a, b, and c show the root loci of individual feedback system represented by equation(11). It is interesting to say that the root loci are not symmetrical about the real axis because the coefficients in equation (11) are complex number. The stable regions of such second-order dynamic system with complex variable were, as precisely discussed in Ref, 10 and 11, given respectively by

$$(a) \text{ for } Q_{\beta} = a_{\beta,R} + ia_{\beta,I} : \\ 1 + K_{\beta} - a_{\beta,I}^2 / K_{\beta}^2 > a_{\beta,R}^2 \quad (14)$$

$$(b) \text{ for } Q_{\beta}^{\cdot} = a_{\beta,R}^{\cdot} + ia_{\beta,I}^{\cdot} : \\ k_{\beta}^{\cdot} > a_{\beta,R}^{\cdot} \quad (15a)$$

and

$$K_{\beta} a_{\beta,R}^2 - 2(K_{\beta} + 1)K_{\beta} a_{\beta,R}^{\cdot} - K_{\beta} a_{\beta,R}^{\cdot 2} + a_{\beta,I}^2 + K_{\beta}^2 (a_{\beta,I}^{\cdot} + K_{\beta} + 1) > 0 \quad (15b)$$

$$(c) \text{ for } Q_{\beta}^{\ddot{}} = a_{\beta,R}^{\ddot{}} + ia_{\beta,I}^{\ddot{}} : \\ K_{\beta} (1 - a_{\beta,R}^{\ddot{}}) + 2a_{\beta,I}^{\ddot{}} > 0 \quad (16a)$$

and

$$K_{\beta}^2 a_{\beta,R}^2 + 2K_{\beta} \{ K_{\beta} (a_{\beta,R}^{\ddot{}} - 1) - (K_{\beta} + 2) \} a_{\beta,I}^{\ddot{}} + K_{\beta}^2 (a_{\beta,R}^{\ddot{}} - 1) (a_{\beta,R}^{\ddot{}} + K_{\beta} + 1) < 0. \quad (16b)$$

From either Fig. 3 or equations (14-16) the stable regions of individual gain of the feedback system are those shown in Fig. 4 a, b, and c. Actually, for the selection of a set of gain combination the following items must be considered: gain margin, natural frequency, response time, allowable overshoot, terminal amplitude, and so on.

The actual pitch link motion must be related to the above feedback gains. This is performed by combining the direct feedback relation,

$$\Delta\theta(t) = c_{\beta} \Delta\beta + c_{\beta}^{\cdot} \Delta\dot{\beta} / \Omega + c_{\beta}^{\ddot{}} \Delta\ddot{\beta} / \Omega^2, \quad (17)$$

with the perturbed relations given by equations (5) and the feedback relations given by equations (9) as follows:

$$\begin{aligned}
c_{\beta}^{\ddot{}} &= b_{\beta}^{\ddot{}} = Q_{\beta}^{\ddot{}} \\
c_{\beta}^{\dot{}} &= b_{\beta}^{\dot{}} = Q_{\beta}^{\dot{}} + 2iQ_{\beta}^{\ddot{}} \\
c_{\beta} &= b_{\beta} = Q_{\beta} - Q_{\beta}^{\ddot{}} + iQ_{\beta}^{\dot{}}
\end{aligned}
\tag{18}$$

By referring to Fig. 3 and Fig. 4 the above equations suggest that the following combination may be a solution for this feedback system:

$$\begin{aligned}
Q_{\beta}^{\ddot{}} &= 0 \\
Q_{\beta}^{\dot{}} &= a_{\beta,R}^{\dot{}} = b_{\beta}^{\dot{}} = K_{\beta}^{\dot{}} - 2\sqrt{K_{\beta}^{\dot{}} + 1} \\
Q_{\beta} &= a_{\beta,R} - i(K_{\beta}^{\dot{}} - 2\sqrt{K_{\beta}^{\dot{}} + 1})
\end{aligned}
\tag{19}$$

$$\begin{aligned}
c_{\beta}^{\ddot{}} &= b_{\beta}^{\ddot{}} = 0 \\
c_{\beta}^{\dot{}} &= b_{\beta}^{\dot{}} = Q_{\beta}^{\dot{}} = K_{\beta}^{\dot{}} - 2\sqrt{K_{\beta}^{\dot{}} + 1} \\
c_{\beta} &= b_{\beta} = Q_{\beta} + iQ_{\beta}^{\dot{}} = a_{\beta,R}
\end{aligned}
\tag{20}$$

where the real part of the Q_{β} , $a_{\beta,R} = b_{\beta}$, may be selected to satisfy the restriction (a) in the stability conditions (12) and (14) and to get the quick response of the system.

4. GUST RESPONSE OF A ROTOR AND A HELICOPTER

Numerical examples of the present calculation have been performed for a rotor fixed in space and a helicopter flying in cruising condition, the detailed dimensions of which are respectively given in Table 1 and 2.

Fig. 5 a,b,c, and d show the time responses of the thrust coefficient, the tilt angles of tip-path plane, and the pitch input for the exemplified rotor penetrating into a sinusoidal vertical gust. The feedback gains were selected as follows:

$$c_{\beta}^{\ddot{}} = 0, \quad c_{\beta}^{\dot{}} = -1.1, \quad \text{and} \quad c_{\beta} = -2, 0.$$

It can be seen from these figures that the deviations of (a) the thrust coefficient and (d) the tilt angles represented by the first harmonic flapping angles from the mean value are appreciably reduced by adopting the above feedback gain if the required pitch angle shown in Fig. 5 c is perfectly followed.

Fig. 6 a, b, c, and d are step responses for the same rotor penetrating into a step gust. The gust alleviation is obvious for such adequate feedback system.

Fig. 7 a and b, and Fig. 8 a and b show the gust responses of the helicopter penetrated into a sinusoidal gust and a step gust respectively. The alleviation effects due to the feedback system are obvious for all quantities related not only to the longitudinal motion but also to the lateral motion.

5. CONCLUSION

A simple analytic method has been proposed to construct a combination of feedback gains in active control system which was designed to alleviate the gust response of the helicopter rotor by sensing the respective blade motion and by making feedback it to the pitch control link of individual blade. The validity of the method was demonstrated in the theoretical calculation based on the local momentum theory for the gust responses of a rotor which is fixed in space and of a helicopter which is flying in cruising condition while they are penetrating into a sinusoidal gust and a step gust.

The deviation of every quantity from a trimmed value is appreciably reduced by adopting such active control system.

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APPENDIX A Coefficients of Equation (4)

$$\begin{aligned}
 A(\psi) &= K_{\beta} + K_{2,a} \mu \sin \psi \\
 B(\psi) &= K_{\beta} + 1 + K_2 \mu \cos \psi + K_3 \mu^2 \sin \psi \cos \psi \\
 C(\psi) &= K_4 \theta_t + K_1 (\theta_0 + \theta_{1c} \cos \psi + \theta_{1s} \sin \psi - \frac{3}{4} \theta_t + 2\mu \theta_t \sin \psi) + K_2 \{ (\theta_0 + \theta_{1c} \cos \psi \\
 &\quad + \theta_{1s} \sin \psi - \frac{3}{4} \theta_t) 2\mu \sin \psi + \mu^2 \theta_t \sin^2 \psi \} + K_3 (\theta_0 + \theta_{1c} \cos \psi + \theta_{1s} \sin \psi \\
 &\quad - \frac{3}{4} \theta_t) \mu^2 \sin^2 \psi - \mu \tan i_s (K_2 + K_3 \mu \sin \psi) - \frac{n_g}{R\Omega^2} K_g + \frac{k_f}{x_f} \beta_0 \\
 &\quad - \left(\frac{v}{R\Omega} \right) \{ (K_1 + K_2 \mu \sin \psi) (K_0 + K_{1c} \cos \psi + K_{1s} \sin \psi) + (1 - \frac{2}{3} K_0) (K_2 + K_3 \mu \sin \psi) \},
 \end{aligned}$$

where,

$$\begin{aligned}
 K_{\beta} &= \frac{1}{x_f^2} (k_f + x_{f,a}^2) - 1 \\
 K_{\beta}^* &= \frac{k_f}{x_f^2} + K_{1,a} \\
 k_f &= k_{\beta} / m_{\beta} R^2 \Omega^2 \\
 k_f^* &= k_{\beta}^* / m_{\beta} R^2 \Omega.
 \end{aligned}$$

APPENDIX B Expression of a Sinusoidal Gust in the Rotor Coordinate System

By referring to Ref.1, the expression of a sinusoidal gust in the rotor coordinate system is given by

$$\begin{aligned}
 w_G(r, t) / w_{G_0} &= \sin(\omega_G t + \kappa X_0) \{ J_0(kx) + 2 \sum_{n=1}^{\infty} (-1)^n J_{2n}(kx) \cos 2n\alpha_{\tau} \} \\
 &\quad - \cos(\omega_G t + \kappa X_0) \{ 2J_1(kx) \cos \alpha_{\tau} \\
 &\quad + 2 \sum_{n=1}^{\infty} (-1)^n J_{2n+1}(kx) \cos(2n+1)\alpha_{\tau} \}, \quad (B-1)
 \end{aligned}$$

where,

$$\begin{aligned}
 k &= \kappa R = \omega_G / \mu \Omega \\
 \omega_G &= \kappa (V \cos \Psi + V_G) \\
 \alpha_{\tau} &= \psi - \Psi \\
 r &= R x.
 \end{aligned} \quad (B-2)$$

By assuming that the wave length of the sinusoidal gust is extremely larger than the rotor radius, the following approximation will be established:

$$\begin{aligned}
 J_0(kx) &= 1 - \frac{k^2 x^2}{4} + \frac{k^4 x^4}{64} - \dots \approx 1 - \frac{k^2 x^2}{4} \\
 J_1(kx) &= \frac{kx}{2} \left(1 - \frac{k^2 x^2}{8} + \frac{k^4 x^4}{32} - \dots \right) \approx \frac{kx}{2} \left(1 - \frac{k^2 x^2}{8} \right) \\
 J_2(kx) &= \frac{k^2 x^2}{4} \left(\frac{1}{2} - \frac{k^2 x^2}{24} + \dots \right) \approx \frac{k^2 x^2}{8} \\
 J_n(kx) &\approx 0 \quad (n \geq 3).
 \end{aligned} \quad (B-3)$$

By using the above approximation, equation (B-1) yields

$$\begin{aligned}
 w_G(r, t) / w_{G_0} &= \sin(\omega_G t + \kappa X_0) \{ J_0(kx) - 2J_2(kx) \cos 2\alpha_{\tau} \} \\
 &\quad - 2J_1(kx) \cos(\omega_G t + \kappa X_0) \cos \alpha_{\tau}.
 \end{aligned}$$

Table 1. Dimensions for a Rotor

R	Rotor radius	, 8.53 m	r_{CG}	C. G. position of blade	, 2.74 m
b	Number of blades	, 4	m_B	Blade mass	, 10.86 kg.sec ² . m ⁻¹
c	Blade chord	, 0.417 m	I	Moment of inertia of blade	, 162.6 kg.m.sec ² .
Ω	Rotor rotational speed	, 23.67 rad./sec.	M_B	Mass moment of blade	, 169.3 kg.m.sec ² .
θ_t	Blade twist angle	, -8 deg.	γ	Lock number	, 8.84
θ_0	Collective pitch angle	, 8 deg.		Wing section	, NACA 0012
θ_{1c}	Longitudinal cyclic pitch angle	, 0 deg.	μ	Advance ratio	, 0.18
θ_{1s}	Lateral cyclic pitch angle	, 0 deg.	W	Gross weight	, 6,353 kg
r_B	Position of flapping hinge	, 0.3 m	ω_G	Gust angular velocity	, 3.14 rad./sec.
r_C	Blade cut off	, 0.594 m	V_G	Gust forward velocity	, 0 m/sec.
			V_{G_0}	Gust amplitude	, 1.80 m/sec.

Table 2. Dimensions for a Exemplified Helicopter

W	Gross weight	, 2994 kg	For Horizontal Wing		
I_{x_B}	Moment of inertia of body	, 339.96 kg*m*s ²	S_H	Wing area	, 1.586 m ²
I_{y_B}	Moment of inertia of body	, 1170.1 kg*m*s ²	b_H	Span	, 2.844 m
I_{z_B}	Moment of inertia of body	, 989.0 kg*m*s ²	c_H	Chord	, 0.558 m
For Main Rotor			A_H	Aspect ratio	, 5.1
R	Rotor radius	, 6.706 m	η_H	Efficiency	, 0.7
b	Number of blades	, 2	For Vertical Wing		
c	Blade chord	, 0.533 m	S_V	Wing area	, 0.883 m ²
θ_t	Blade twist	, -10 deg.	b_V	Span	, 1.036 m
Ω	Rotor rotational speed	, 33.9 rad./sec.	c_V	Chord	, 0.884 m
m_B	Blade mass	, 8.837 kg*s ² *m ⁻¹	A_V	Aspect ratio	, 1.22
I_B	Moment of inertia of blade	, 118.4 kg*m*s ²	η_V	Efficiency	, 0.9
i_s	Inclination of rotor shaft	, 3 deg.	Inputs		
γ	Lock number	, 6.526	θ_0	Collective pitch angle	, 6.4 deg.
σ	Solidity	, 0.0506	θ_{1c}	Longitudinal cyclic pitch angle	, -0.2 deg.
For Tail Rotor			θ_{1s}	Lateral cyclic pitch angle	, -1.45 deg.
R_T	Rotor radius	, 1.3 m	ϕ	Roll angle of body	, 0 deg.
b_T	Number of blades	, 2	θ	Pitch angle of body	, -3.5 deg.
c_T	Blade chord	, 0.214 m	ψ	Yaw angle of body	, 0 deg.
θ_{tT}	Blade twist	, 0 deg.	γ	Flight path angle	, 0 deg.
Ω_T	Rotor rotational speed	, 173 rad./sec.	μ	Advance ratio	, 0.2
m_{BT}	Blade mass	, 0.23 kg*s ² *m ⁻¹	ω_G	Gust frequency	, 3.14 rad./sec.
I_{BT}	Moment of inertia of blade	, 0.128 kg*m*s ²	v_{G_0}	Gust amplitude	, 1.8 m/sec.
δ_{jT}	δ_3 - angle	, 45 deg.	c_{β}	Feedback gain of $\Delta\beta$, -1.0
γ_T	Lock number	, 3.35	c_{β}^2	Feedback gain of $\Delta\beta^2$, -1.1
σ_T	Solidity	, 0.105	c_{β}^3	Feedback gain of $\Delta\beta^3$, 0

- (a) STATIONARY COORDINATE SYSTEM , (X, Y, Z)
- (b) HUB COORDINATE SYSTEM , (X_H, Y_H, Z_H)
- (c) ROTOR COORDINATE SYSTEM , (X_R, Y_R, Z_R)

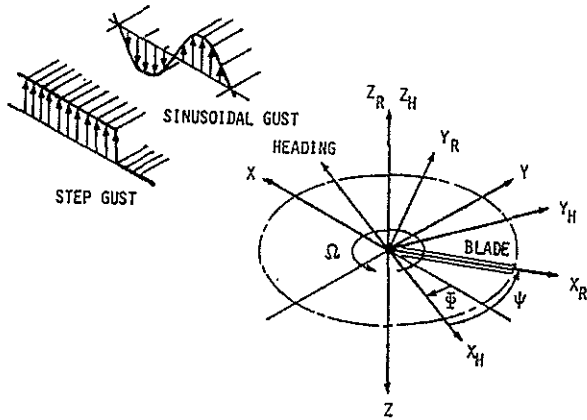


Figure 1 Gust shapes and the related coordinate systems.

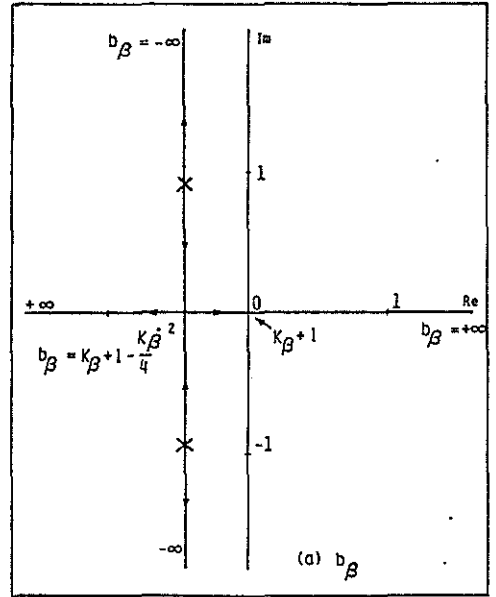
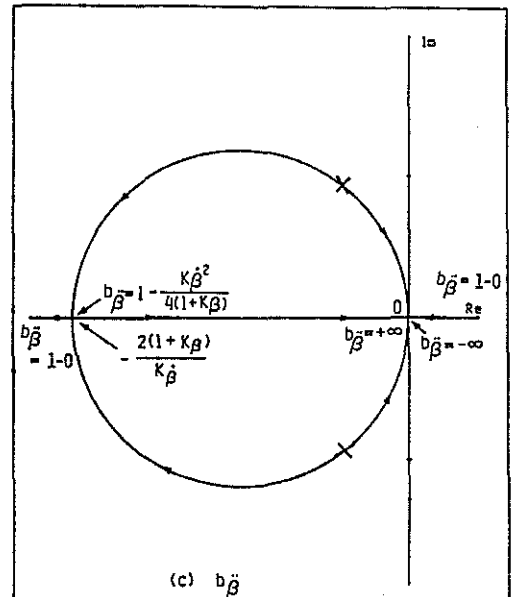
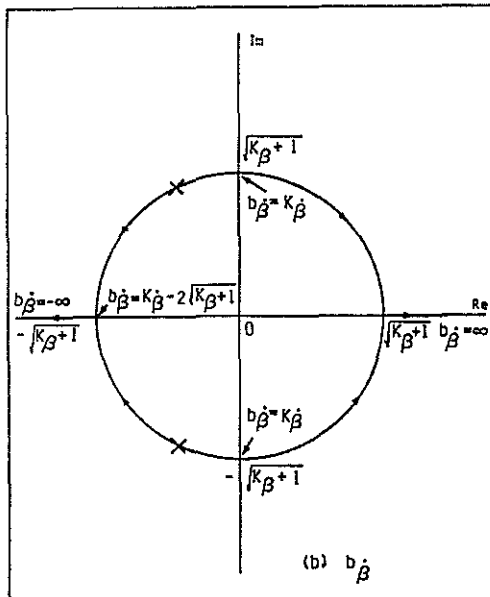


Figure 2 Root loci for the individual feedback system of the coning angle.



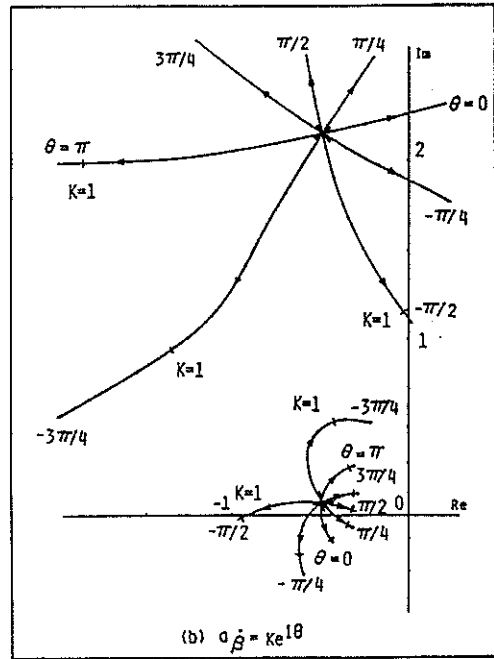
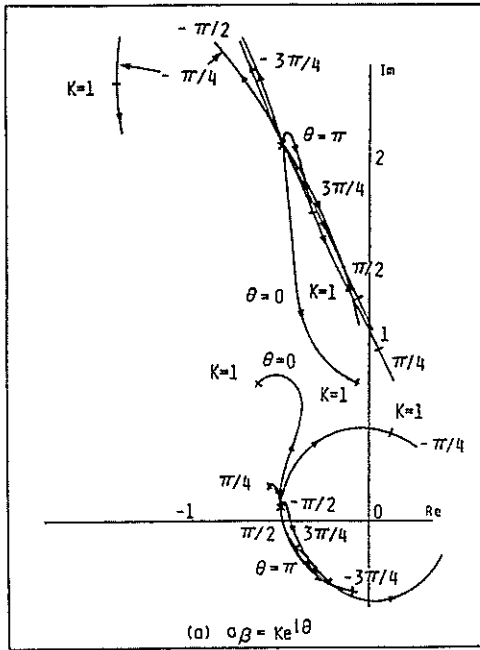


Figure 3 Root loci for the individual feedback system of the complex tilt angle.

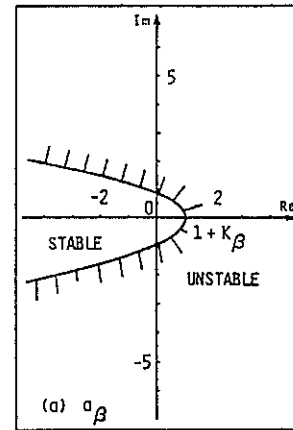
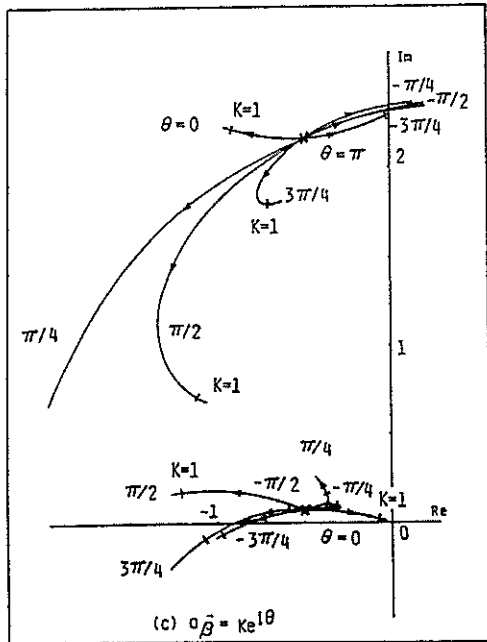
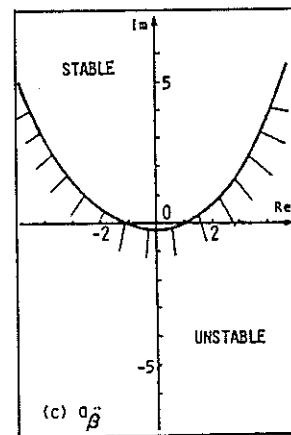
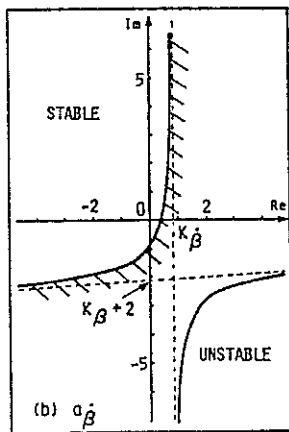


Figure 4 Stable regions of the individual feedback gain for the complex tilt angle.



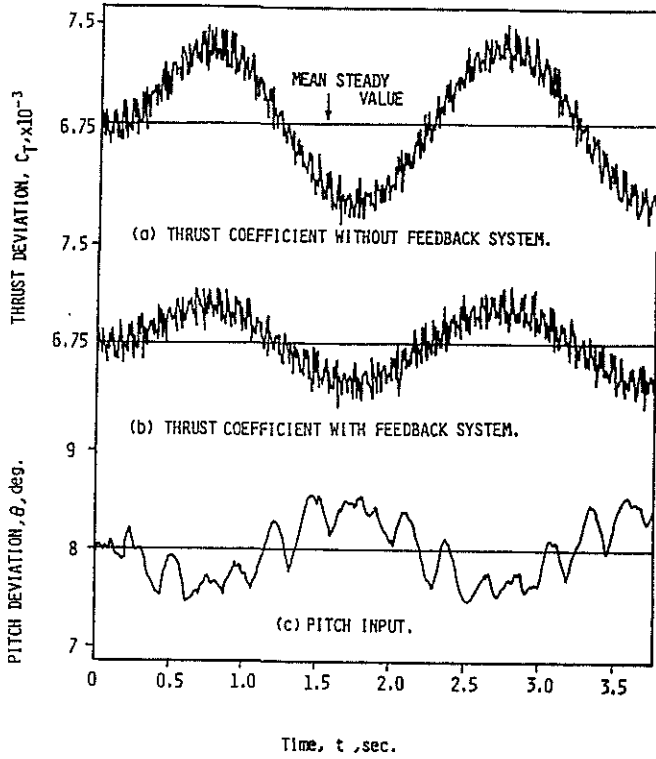
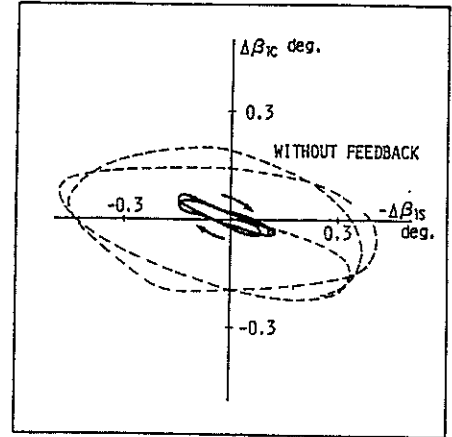


Figure 5 Time responses for a sinusoidal gust.

$$(w_{G0} = 1.8 \text{ m/sec.}, c_{\beta} = -2.0, c_{\dot{\beta}} = -1.1, c_{\ddot{\beta}} = 0, \mu = 0.18)$$



(d) TIP PATH PLANE

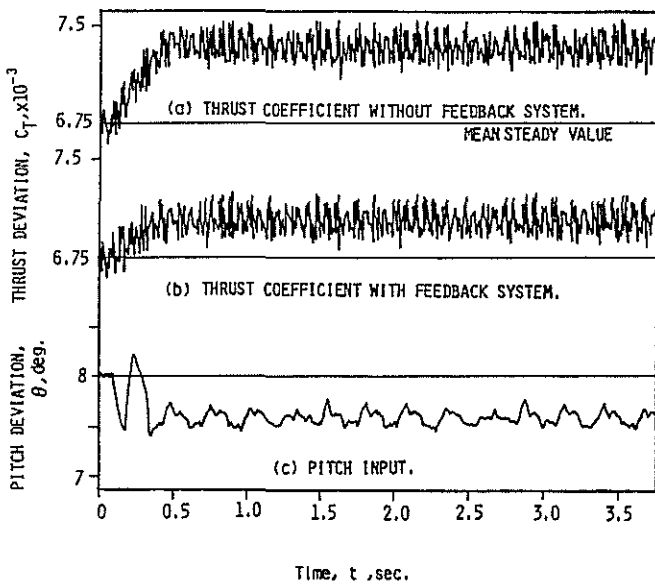
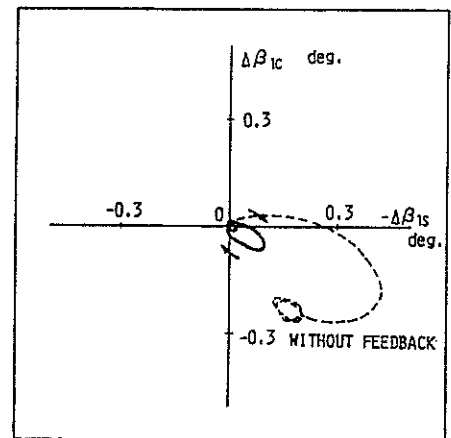
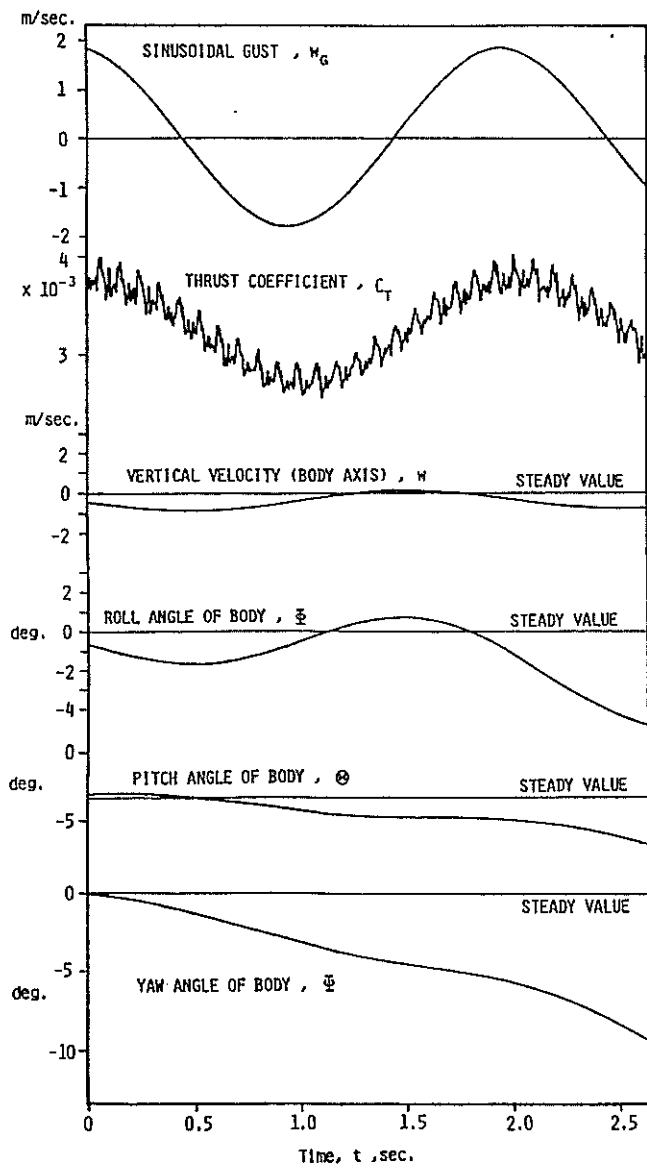


Figure 6 Time responses for a step gust.

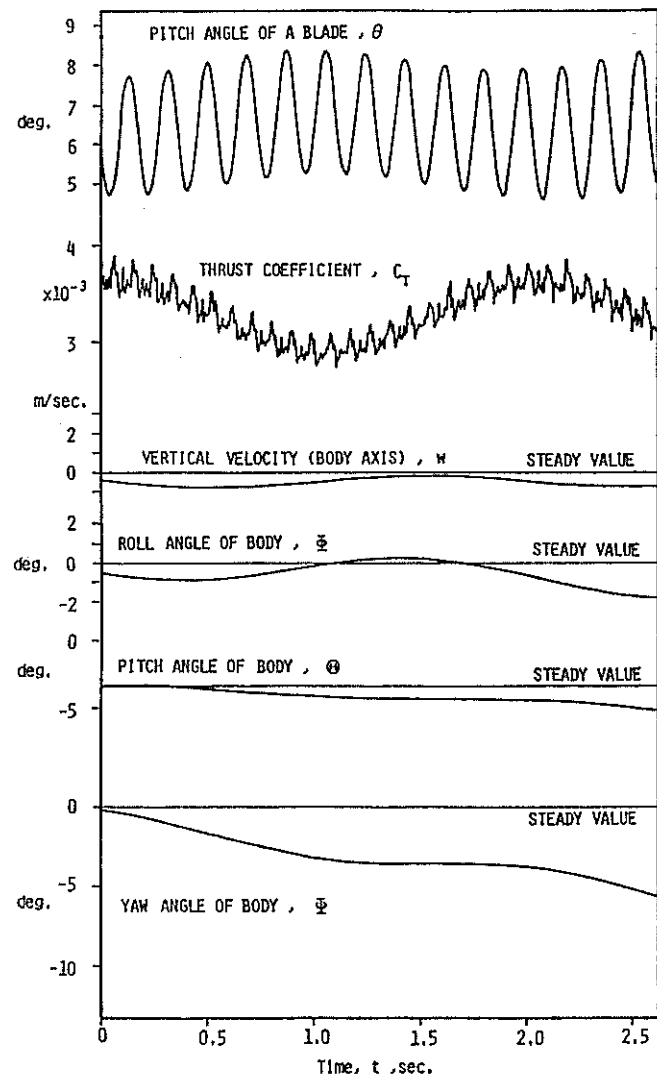
$$(w_{G0} = 1.8 \text{ m/sec.}, c_{\beta} = -2.0, c_{\dot{\beta}} = -1.1, c_{\ddot{\beta}} = 0, \mu = 0.18)$$



(d) TIP PATH PLANE



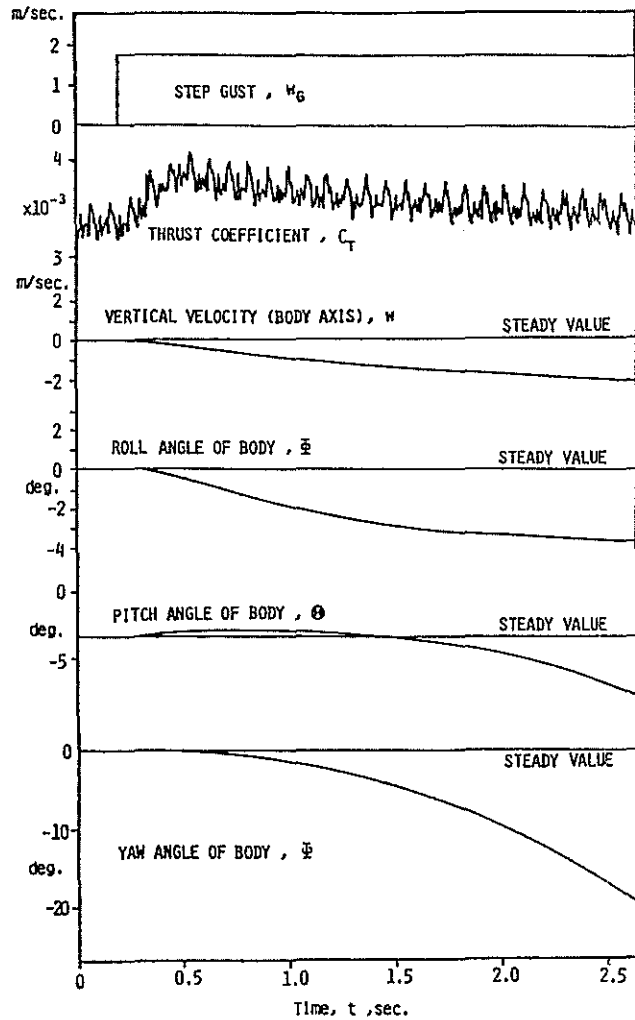
(a) WITHOUT FEEDBACK CONTROL SYSTEM.



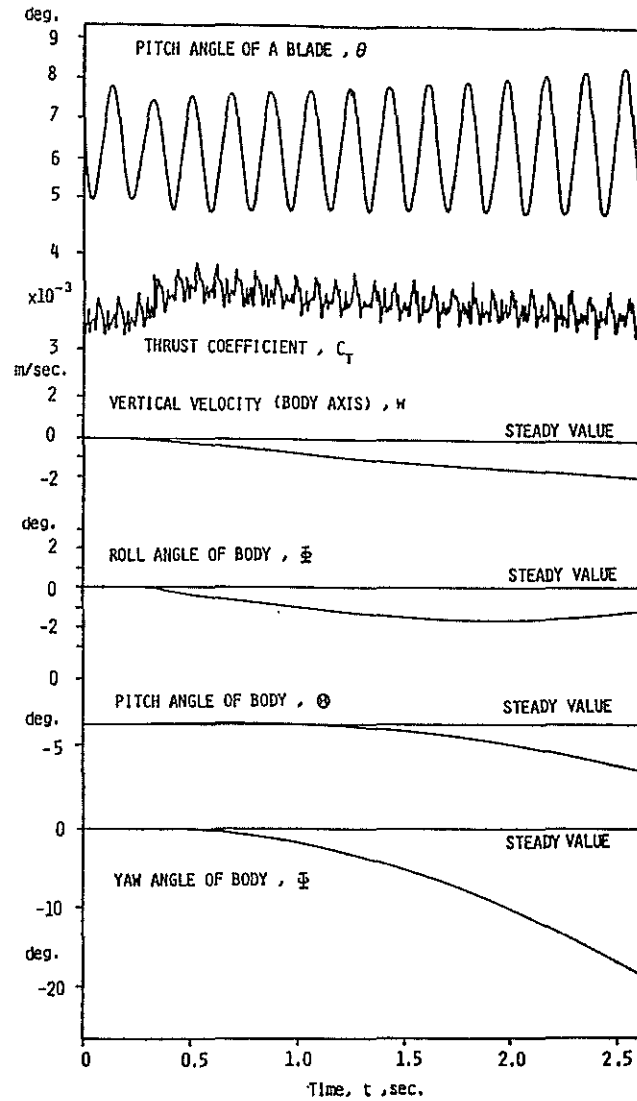
(b) WITH FEEDBACK CONTROL SYSTEM.

Figure 7 Examples of the reduction due to feedback control system.

$$(w_{G0} = 1.8 \text{ m/sec.}, \omega_G = 3.14 \text{ rad/sec.}, c_{\beta} = -1.0, c_{\beta}^* = -1.1, c_{\beta}^{**} = 0)$$



(a) WITHOUT FEEDBACK CONTROL SYSTEM.



(b) WITH FEEDBACK CONTROL SYSTEM.

Figure 8 Examples of the reduction due to feedback control system.

$$(w_{G0} = 1.8 \text{ m/sec.}, c_{\beta} = -1.0, c_{\beta}^{\dot{\theta}} = -1.1, c_{\beta}^{\ddot{\theta}} = 0)$$